

MODULE II : MANEUVERABILITY

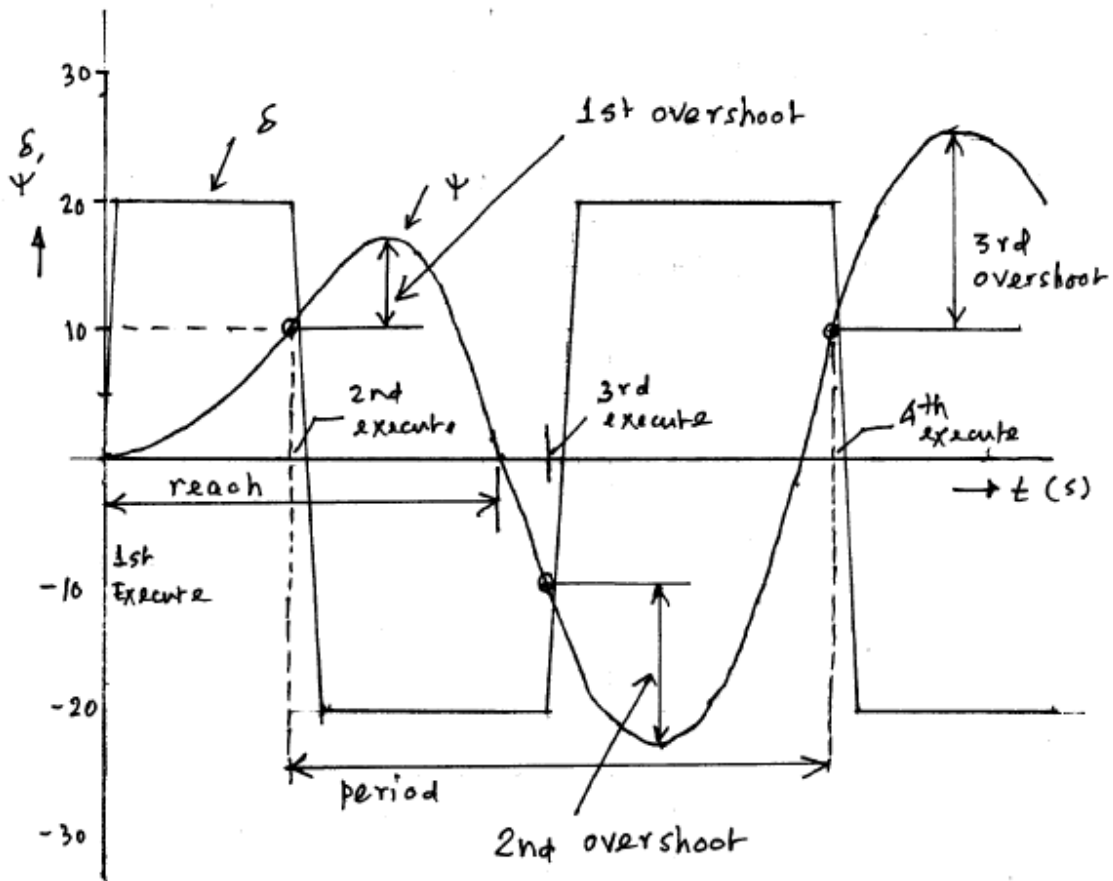
Topic : Definitive Maneuvers

Question 1

Draw a schematic diagram of a 20/10 zigzag maneuver and identify all the terms.

Answer:

The figure is shown below and the terms like (time of) 1st, 2nd, 3rd executes, (time to) reach, period, 1st and 2nd overshoot etc. are explained on the diagram.



δ : rudder angle, ψ : heading .

Question 2

(i) The sway and yaw maneuvering equations with rudder working are as follows:

$$\begin{aligned} -Y_v v + (m - Y_{\dot{v}}) \dot{v} - (Y_r - mU)r - Y_r \dot{r} &= Y_{\delta} \delta \\ -N_v v - N_{\dot{v}} \dot{v} - N_r r + (I_z - N_{\dot{r}}) \dot{r} &= N_{\delta} \delta \end{aligned}$$

From this, show that the steady turning radius for a given ship at a given rudder angle is independent of forward speed but its yaw rate is in proportion to forward speed.

(ii) For a ship of $L = 110\text{m}$, $B = 18\text{m}$, $T = 4.1\text{m}$, $C_B = 0.68$, the hydrodynamic and rudder derivatives are as follows:

$$\begin{aligned} Y'_v &= -9.65 \times 10^{-3} ; Y'_r = 2.14 \times 10^{-3} ; N'_v = -2.57 \times 10^{-3} ; N'_r = -1.44 \times 10^{-3} \\ Y'_{\delta} &= -1.0 \times 10^{-3} ; N'_{\delta} = 0.5 \times 10^{-3} \end{aligned}$$

Find its turning radius, drift angle and yaw rate for 16 knots at 35 deg. rudder.

Answer:

(i)

During steady turning phase, the time derivatives are all zero, and therefore the sway and yaw equations reduces to (obtained by making $\dot{v} = \dot{r} = 0$) :

$$\begin{aligned} -Y_v v - (Y_r - mU)r &= Y_{\delta} \delta \\ -N_v v - N_r r &= N_{\delta} \delta \end{aligned}$$

In non-dimensional form:

$$\begin{aligned} -Y'_v v' - (Y'_r - m')r' &= Y'_{\delta} \delta \\ -N'_v v' - N'_r r' &= N'_{\delta} \delta \end{aligned}$$

From this we get by solving for v' and r' :

$$v' = \delta \left[\frac{N'_{\delta}(Y'_r - m') - Y'_{\delta}N'_r}{Y'_v N'_r - N'_v(Y'_r - m')} \right], \quad r' = \delta \left[\frac{N'_v Y'_{\delta} - Y'_v N'_{\delta}}{Y'_v N'_r - N'_v(Y'_r - m')} \right]$$

We have (from definition of non-dimensionalization), $r' = rL/V$. Also, $V = rR$.

Thus, $r' = rL/V = rL/rR = L/R$

We then get,

$$r = r' \frac{V}{L} = \delta \frac{V}{L} \left[\frac{N'_v Y'_\delta - Y'_v N'_\delta}{Y'_v N'_r - N'_v (Y'_r - m')} \right]; \quad \frac{R}{L} = \frac{1}{r'} = \frac{1}{\delta} \left[\frac{Y'_v N'_r - N'_v (Y'_r - m')}{N'_v Y'_\delta - Y'_v N'_\delta} \right]$$

This shows that for a given ship, yaw rate $r \propto V, \delta$, and the turning radius $R \propto \frac{1}{\delta}$ but independent of V .

From the definition of non-dimensional sway, we have $v' = \frac{v}{V} = \frac{V \sin(-\beta)}{V} \approx -\beta$.

Thus drift angle is given by $-v'$, the expression of which is given above. It is seen that drift angle is in proportion to δ but independent of V .

(ii)

$$\text{Here } m' = \frac{m}{0.5\rho L^3} = \frac{\rho L B T C_B}{0.5\rho L^3} = \frac{2 B T C_B}{L^2} = \frac{(2)(18)(4.1)(0.68)}{180^2} = 3.098 \times 10^{-3}$$

Turning radius

$$R = \frac{L}{\delta} \left[\frac{Y'_v N'_r - N'_v (Y'_r - m')}{N'_v Y'_\delta - Y'_v N'_\delta} \right] = \frac{110}{35 \times \pi / 180} \left[\frac{(-9.65)(-1.44) - (-2.57)(2.14 - 3.098)}{(-2.57)(-1) - (-9.65)(0.5)} \right]$$

$$= 279 \text{ m}$$

Yaw rate

$$r = \delta \frac{V}{L} \left[\frac{N'_v Y'_\delta - Y'_v N'_\delta}{Y'_v N'_r - N'_v (Y'_r - m')} \right] = 1.69 \text{ deg/sec}$$

Drift angle

$$\beta = -\delta \left[\frac{N'_\delta (Y'_r - m') - Y'_\delta N'_r}{Y'_v N'_r - N'_v (Y'_r - m')} \right] = 9.61 \text{ deg}$$

Question 3

(i)

Find an approximate expression for heel angle during steady turn.

(ii)

A vessel turns in a radius of 300 m. at a speed of 16 knots under the action of rudder force of 2 MN. If the draft of the ship is 8m, KG is 9m and GM is 3m, find the approximate angle of heel during the steady turn

Answer:

(i)

During steady turn, let F_h and F_r are the hull and rudder forces respectively.

Referring to the figure below, by equating forces, we have $F_h - F_r = \frac{\Delta V^2}{Rg}$ where Δ is mass, R is turning radius, and V is speed during turn.

If F_h and F_r acts at the points H and E respectively as shown in the diagram, the heeling moment is given by

$$\begin{aligned} & (F_h - F_r)(KG) + F_r(KH) - F_h(KE) \\ &= (F_h - F_r)(KG - KE) + F_r(KH - KE) \\ &= (F_h - F_r)(GE) - F_r(EH) \end{aligned}$$

(K represent point on keel)

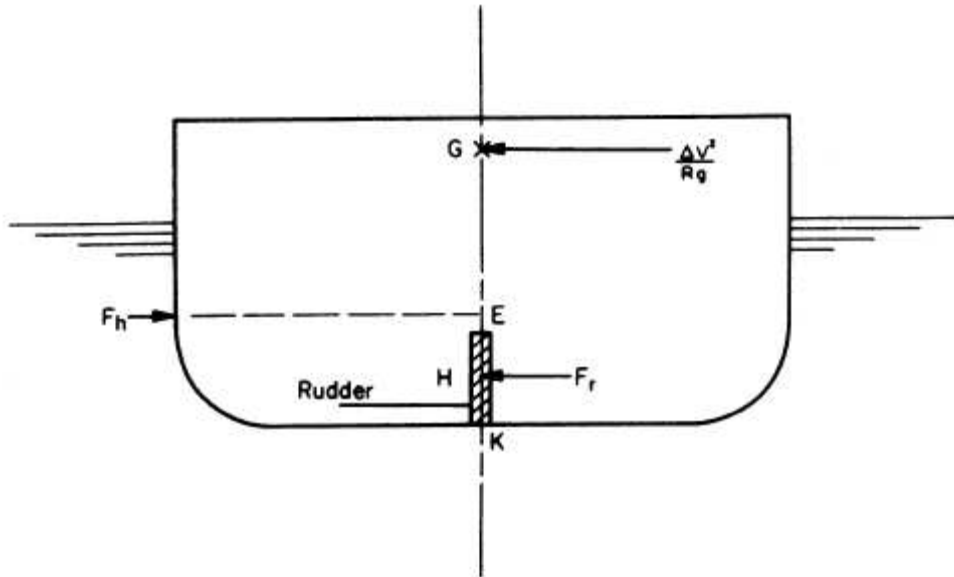
For most ships, both E and H are approximately same and at the half draft level. With this approximation, heeling moment becomes $(F_h - F_r)(GE)$

If the heel angle is α , we have

$$\Delta GM \sin \alpha = (F_h - F_r)(GE) = \frac{\Delta V^2}{Rg} (GE)$$

Thus for small α for which $\sin \alpha \approx \alpha$, we get the heel angle as:

$$\alpha = \frac{V^2}{Rg} \frac{GE}{GM}$$



(ii)

For this problem, $V = 16$ knots, and $R = 450$ m.

Assuming that the rudder force acts at half draft level, we have $GE = KG - T/2 = 9 - 8/2 = 5$ m

$$\text{Thus, } \alpha = \frac{(16 \times 0.5144)^2}{(300)(9.8)} \frac{5}{3} = 0.0384 \text{ rad} = 2.2 \text{ deg.}$$