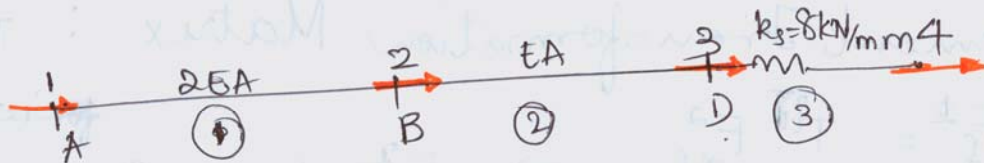
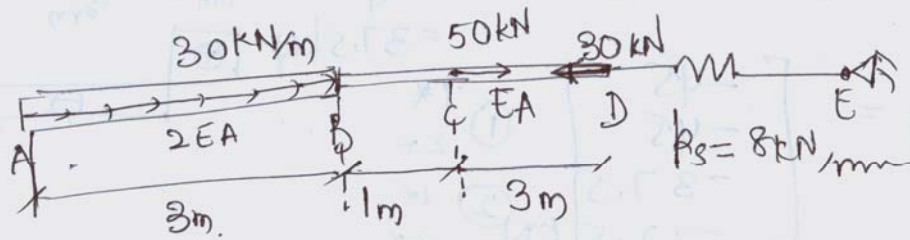


Tutorial 4 - Axial Elements (Matrix Methods) CE3310

$$EA = 100 \times 30 \text{ KN} = 3000 \text{ KN}$$

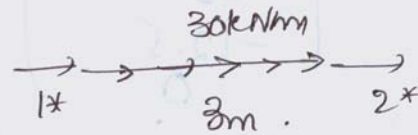
M.S. SARVANI
CE09B030

1. (1)



Fixed End Forces:

$$F_{*f}^1 = \begin{Bmatrix} -45 \\ -45 \end{Bmatrix} \text{ KN}$$



$$\frac{90}{2} = 45 \text{ KN}$$

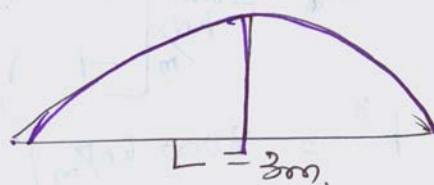
Member Axial force Diagram:



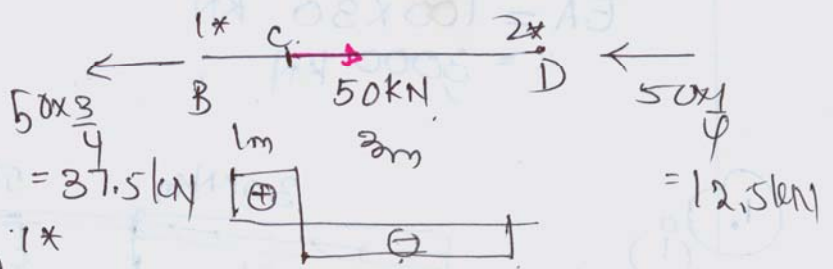
$$\frac{dU^*}{dx} = \frac{N(x^*)}{EA}$$

$$U^* = \frac{1}{EA} \int_0^L (45 - 30x^*) dx = 45x^* - 15(x^*)^2$$

$$U_{max}^* = \frac{1}{EA} \left(45 \times \frac{1}{2} - \frac{15}{4} \right) = \frac{18.75}{EA}$$



$$F_{*f}^2 = \begin{bmatrix} -37.5 \\ -12.5 \end{bmatrix} \text{ kN}$$



$$F_{*f}^1 = \begin{bmatrix} -45 \\ -45 \\ -37.5 \\ -12.5 \end{bmatrix} \text{ kN}$$

① 1*
② 2*

Element Transformation Matrix : $T^i = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} = \hat{I}$

$$F_f^2 = T^{iT} F_{*f}^2$$

$$\Rightarrow F_f^2 = \begin{bmatrix} -37.5 \\ -12.5 \end{bmatrix}$$

$$T^1 = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$T^2 = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$F_f^1 = \begin{bmatrix} -45 \\ -45 \end{bmatrix}$$

$$\Rightarrow F_f = \begin{bmatrix} -45 \\ -45 - 37.5 \\ -12.5 \\ 0 \end{bmatrix} = \begin{bmatrix} -45 \\ -82.5 \\ -12.5 \\ 0 \end{bmatrix} \text{ kN}$$

Element Stiffness Matrix

$$k_{*}^1 = \frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k_{*}^2 = \frac{EA}{4} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$\Rightarrow k^1 = \frac{2EA}{3} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k_{*}^3 = 8 \text{ kN/mm} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$T^{iT} k_{*} T^i$$

$$k_{*}^1 = \frac{2000}{m} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k^2 = 750 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k^3 = 8000 \text{ kN/m} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k = \begin{bmatrix} \textcircled{1} & \textcircled{2} & \textcircled{3} & \textcircled{4} \\ 2000 & -2000 & 0 & 0 \\ -2000 & 2750 & -750 & 0 \\ 0 & -750 & 8750 & -8000 \\ 0 & 0 & -8000 & 8000 \end{bmatrix} \begin{matrix} \textcircled{1} \\ \textcircled{2} \\ \textcircled{3} \\ \textcircled{4} \end{matrix} \quad \text{kN/m}$$

$$= \begin{bmatrix} \textcircled{1} & \textcircled{2} & \textcircled{3} & \textcircled{4} \\ 2000 & -2000 & 0 & 0 \\ -2000 & 2750 & -750 & 0 \\ 0 & -750 & 8750 & -8000 \\ 0 & 0 & -8000 & 8000 \end{bmatrix} \quad \text{kN/m}$$

Active coordinates $\textcircled{1}, \textcircled{2}, \textcircled{3}$, restrained $\textcircled{4}$

Global force vector: $\{F\} = \begin{Bmatrix} 0 \\ 0 \\ -30 \\ F_4 = ? \end{Bmatrix}$; $D = \begin{Bmatrix} D_1 \\ D_2 \\ D_3 \\ D_4 = 0 \end{Bmatrix}$

(Load Input)

$$\{F\} = \{F\} = k \{D\}$$

$$\begin{Bmatrix} 0 \\ 0 \\ -30 \\ F_4 \end{Bmatrix} = \begin{bmatrix} 2000 & -2000 & 0 & 0 \\ -2000 & 2750 & -750 & 0 \\ 0 & -750 & 8750 & -8000 \\ 0 & 0 & -8000 & 8000 \end{bmatrix} \begin{Bmatrix} D_1 \\ D_2 \\ D_3 \\ D_4 = 0 \end{Bmatrix}$$

$$\Rightarrow F_A = K_{AA} D_A + (K_{AR} D_R) \rightarrow 0 = K_{AA} D_A$$

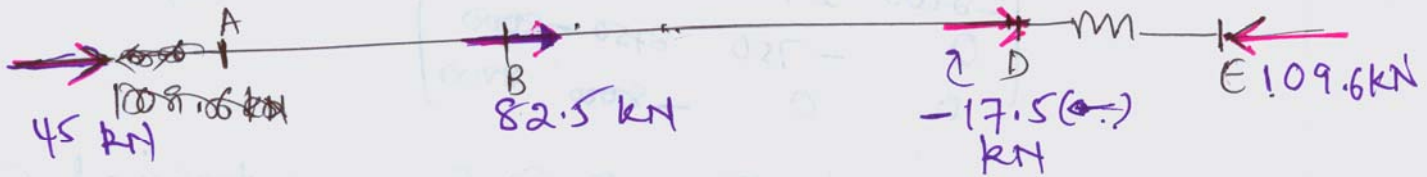
$$\Rightarrow \begin{Bmatrix} +45 \\ 82.5 \\ -17.5 \end{Bmatrix} = K_{AA} \{D_A\}$$

$$\{D_A\} = K_{AA}^{-1} \{F_A\} = \{0.20625\}$$

$$F_y = -8000 D_3 \quad \text{kN/m} \times \text{m}$$

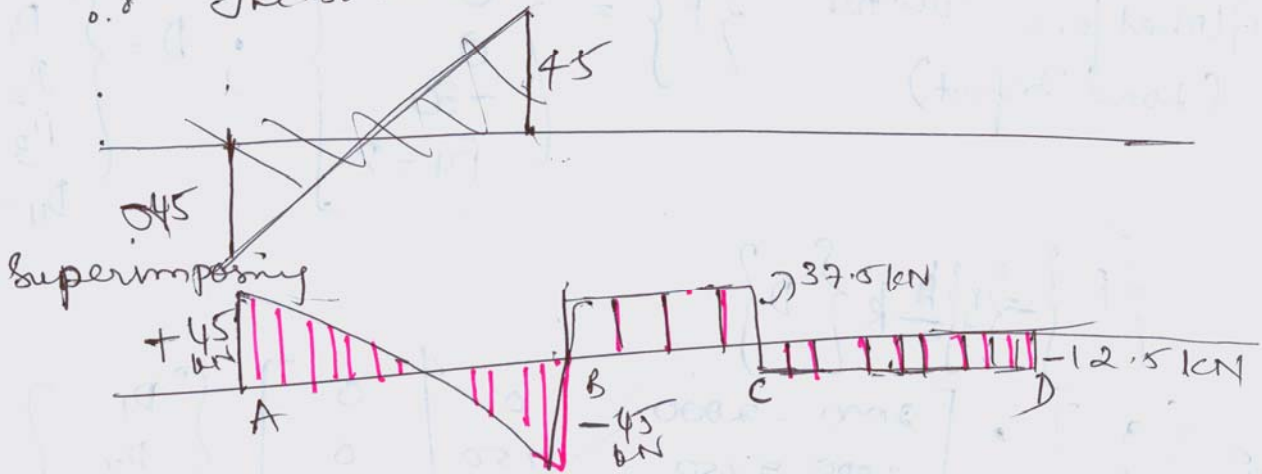
$$= -8000 \times 0.0137 = -109.6 \text{ kN}$$

$$F_y = -109.6 \text{ kN}$$

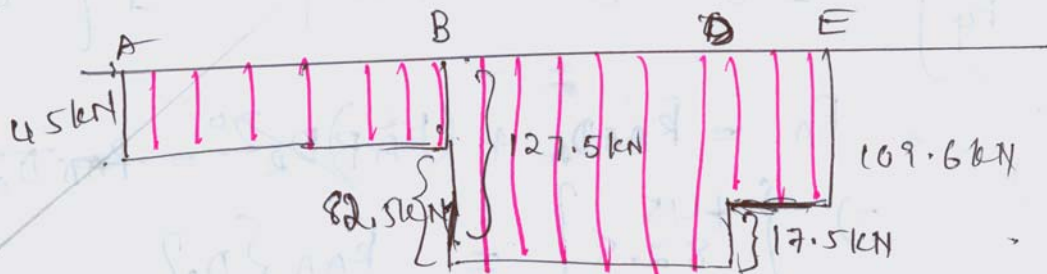


$\Sigma F_x = 0$ is satisfied

\therefore The solution is correct



and



Axial force diagram:

B C C D E

(ii) Displacement Transformation Matrix T^D

$$D^* = T^D D$$

$$T^D = \begin{matrix} \text{①} & \text{②} & \text{③} & \text{④} \\ \begin{matrix} 1^* \\ 2^* \end{matrix} & \begin{matrix} 1 \\ 0 \end{matrix} & \begin{matrix} 0 \\ 1 \end{matrix} & \begin{matrix} 0 \\ 0 \end{matrix} \\ \begin{matrix} 1^* \\ 2^* \end{matrix} & \begin{matrix} 0 \\ 0 \end{matrix} & \begin{matrix} 1 \\ 0 \end{matrix} & \begin{matrix} 0 \\ 1 \end{matrix} \\ \begin{matrix} 1^* \\ 2^* \end{matrix} & \begin{matrix} 0 \\ 0 \end{matrix} & \begin{matrix} 1 \\ 0 \end{matrix} & \begin{matrix} 0 \\ 1 \end{matrix} \end{matrix}$$



$$k = T_D^T k^* T_D$$

where $k^* = \begin{bmatrix} k_1^* & 0 & 0 \\ 0 & k_2^* & 0 \\ 0 & 0 & k_3^* \end{bmatrix}$

(from previous problems)

$$\Rightarrow k = \begin{bmatrix} 2000 & -2000 & 0 & 0 \\ -2000 & 2750 & -750 & 0 \\ 0 & -750 & 8750 & -8000 \\ 0 & 0 & -8000 & 8000 \end{bmatrix} \text{ kN/m.}$$

$$\{F - F_f\} = k D$$

$$F_f = \begin{matrix} \text{①} \\ \text{②} \end{matrix} \begin{matrix} 1^* \\ 2^* \end{matrix} \left\{ \begin{matrix} -45 \\ -45 \\ -37.5 \\ -12.5 \end{matrix} \right\} \text{ kN.}$$

$$F_f = T_D^T F_{xf}$$

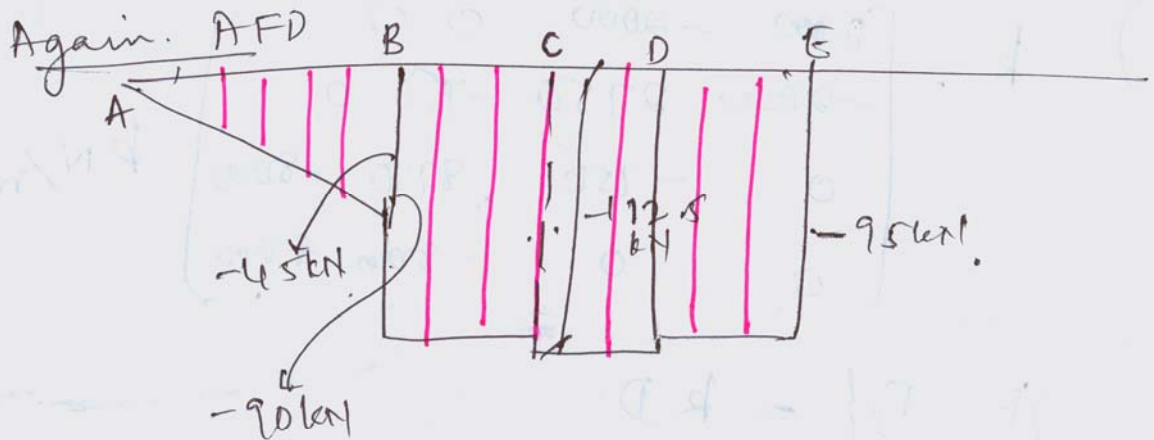
$$F_f = \begin{Bmatrix} -45 \\ -82.5 \\ -12.5 \\ 0 \end{Bmatrix} \text{ kN}$$

$$F = \begin{Bmatrix} 0 \\ 0 \\ -30 \\ F_4 \end{Bmatrix}$$

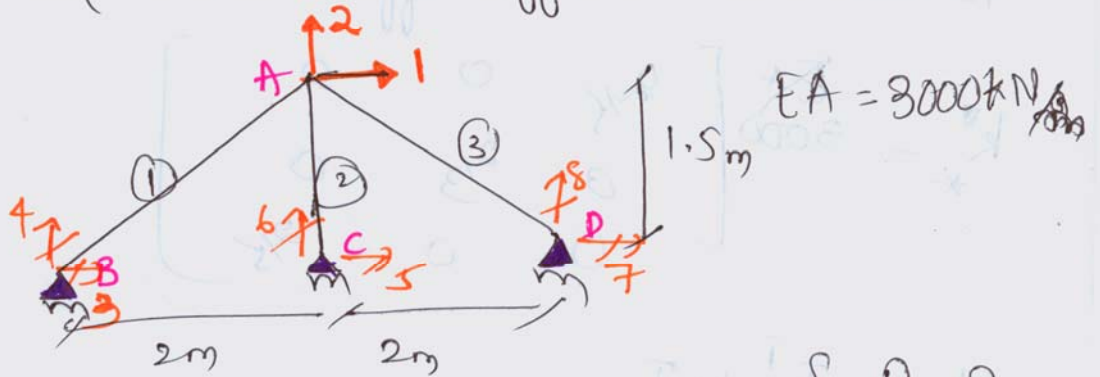
$$F - F_f = k D \quad (\text{similar to (i) Part})$$

$$\Rightarrow \begin{Bmatrix} D \\ A \end{Bmatrix} = \begin{Bmatrix} 206.2 \text{ mm} \\ 183.7 \text{ mm} \\ 13.7 \text{ mm} \end{Bmatrix}$$

$$\Rightarrow F_4 = -109.6 \text{ kN}$$



Q.2) Reduced Element Stiffness Method.



Bar 2 : short by 5mm $e_0 = \begin{Bmatrix} 0 \\ -5 \\ 0 \end{Bmatrix}$ mm.
 (initial displacement vector)

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \begin{Bmatrix} 50 \\ -25 \end{Bmatrix} \text{ kN}$$

SN _o .	Start Node	End Node	θ_x^i	$\cos \theta_x^i$	$\sin \theta_x^i$	$L_i = \frac{1.5}{\sin}$
1	B	A	$\tan^{-1} \frac{3}{4}$	$\frac{4}{5}$	$\frac{3}{5}$	2.5m
2	C	A	90	0	1	1.5m
3	D	A	$\cos^{-1}(-\frac{4}{5})$	$-\frac{4}{5}$	$\frac{3}{5}$	2.5m

$$T_D^r = \begin{bmatrix} \textcircled{3} & \textcircled{4} & \textcircled{1} & \textcircled{2} \\ -\frac{4}{5} & -\frac{3}{5} & \frac{4}{5} & \frac{3}{5} \\ \textcircled{5} & \textcircled{6} & \textcircled{1} & \textcircled{2} \\ 0 & -1 & 0 & 0 \end{bmatrix}$$

$$T_D^i = [-\cos \theta \quad -\sin \theta \quad 1 \quad 0]$$

Reduced element stiffness matrix :

$$k^* = 3000 \begin{bmatrix} 2/5 & 0 & 0 \\ 0 & 2/3 & 0 \\ 0 & 0 & 2/5 \end{bmatrix}$$

$$k = T_D^T k^* T_D$$

$$k^* T_D = k^* \begin{bmatrix} T_{DA} & | & T_{DR} \end{bmatrix}$$

$\begin{matrix} 3 \times 3 & & 3 \times 2 \\ & & 3 \times 6 \end{matrix}$

$$T_D^T k^* T_D = T_D^T \begin{bmatrix} k^* T_{DA} & | & k^* T_{DR} \end{bmatrix}$$

$$= \begin{bmatrix} T_{DA}^T & \\ & T_{DR}^T \end{bmatrix} \begin{bmatrix} k^* T_{DA} & | & k^* T_{DR} \end{bmatrix}$$

$$= \begin{bmatrix} T_{DA}^T k^* T_{DA} & | & T_{DA}^T k^* T_{DR} \\ \hline T_{DR}^T k^* T_{DA} & | & T_{DR}^T k^* T_{DR} \end{bmatrix}$$

$$= \begin{bmatrix} k_{AA} & | & k_{AR} \\ \hline k_{RA} & | & k_{RR} \end{bmatrix}$$

$$\Rightarrow k_{AA} = T_{DA}^T k^* T_{DA}$$

$$T_D = \begin{bmatrix} 1 & 2 & 3 & 4 & 5 & 6 & 7 & 8 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$T_{DA} = \begin{bmatrix} \frac{4}{5} & \frac{3}{5} \\ 0 & 1 \\ -\frac{4}{5} & \frac{3}{5} \end{bmatrix}$$

$$\begin{aligned} \Rightarrow k_{AA} &= T_{DA}^T k_* T_{DA} \\ &= \begin{bmatrix} 1536 & 0 \\ 0 & 2864 \end{bmatrix} \text{ kN/m} \\ &= 3000 \begin{bmatrix} 0.512 & 0 \\ 0 & 0.955 \end{bmatrix} \text{ kN/m} \end{aligned}$$

Input Load Data :

$$\text{Active } \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \begin{Bmatrix} 50 \\ -25 \end{Bmatrix} \text{ kN.}$$

Bar 2 is too short by 5mm, so

$$F_{f*}^2 = -\frac{EA}{1.5} \times (-5 \times 10^{-3}) = \frac{0.01}{3} \times 3000 = 10 \text{ kN.}$$

$$F_f^* = \begin{Bmatrix} 0 \\ +10 \\ 0 \end{Bmatrix} \text{ kN} \Rightarrow F_f^T = \begin{Bmatrix} 0 & 10 & 0 & 0 & 0 & -10 & 0 & 0 \end{Bmatrix} \text{ kN.}$$

Using $F_f^T = T_D^T F_f^*$
Equilibrium

$$F - F_f = k_D D$$

$$\begin{Bmatrix} F_A - F_{fA} \\ F_R - F_{fR} \end{Bmatrix} = \begin{bmatrix} k_{AA} & k_{AR} \\ k_{RA} & k_{RR} \end{bmatrix} \begin{Bmatrix} D_A \\ D_R = 0 \end{Bmatrix}$$

$$\Rightarrow D_A = [k_{AD}] \{F_A - E_{FA}\} \quad F_{FA} = \begin{Bmatrix} 0 \\ 10 \end{Bmatrix} \text{ kN}$$

$$= \begin{bmatrix} 1536 & 0 \\ 0 & 2684 \end{bmatrix} \begin{Bmatrix} 50 \\ -25 - 10 = -35 \end{Bmatrix}$$

$$\{D_A\} = \begin{Bmatrix} D_1 \\ D_2 \end{Bmatrix} = \begin{Bmatrix} 0.0325 \text{ m} \\ -0.013 \text{ m} \end{Bmatrix} = \begin{Bmatrix} 32.5 \text{ mm} \\ -13.0 \text{ mm} \end{Bmatrix} \text{ Ans.}$$

Bar forces $F_A = k_{AD} D_A$

$$F_A = k_{AD} D_A$$

$$= k_{AD} T_D D$$

$$= 3000 \begin{bmatrix} 2/5 & 0 & 0 \\ 0 & 2/3 & 0 \\ 0 & 0 & 2/5 \end{bmatrix} \begin{bmatrix} -4/5 \\ T_D \\ 3/5 \end{bmatrix} D$$

$$= k_{AD} \begin{Bmatrix} T_{DA} \\ T_{DR} \end{Bmatrix} \begin{Bmatrix} D_A \\ D_R \end{Bmatrix}$$

$$= k_{AD} [T_{DA} D_A + T_{DR} D_R]$$

$$= k_{AD} T_{DA} D_A$$

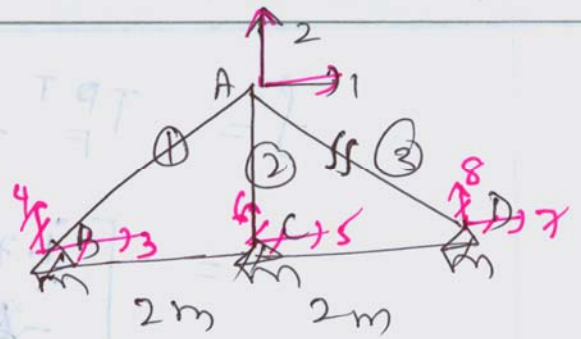
Member

$$F_A = 1 \times [21.874]$$

Q.3

Bar (3) is redundant.

$$\text{Displacement } \bar{x} = \begin{Bmatrix} 0 \\ -5 \text{ mm} \\ 0 \end{Bmatrix}$$



$$F_{\text{Prop}} \text{ Load Input} = \begin{Bmatrix} F_A \end{Bmatrix} = \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \begin{Bmatrix} +50 \\ -25 \end{Bmatrix} \text{ kN}$$

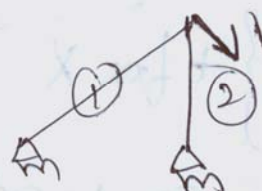
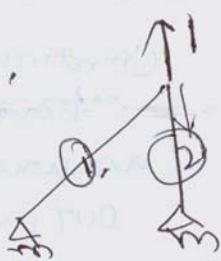
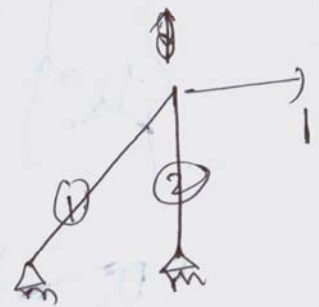
Flexibility Matrix

$$f^* = \begin{bmatrix} 5/2EA & 0 & 0 \\ 0 & 3/2EA & 0 \\ 0 & 0 & 5/2EA \end{bmatrix} \text{ (kN/m)}^{-1}$$

$$f^* = 10^{-4} \begin{bmatrix} 8.33 & 0 & 0 \\ 0 & 5 & 0 \\ 0 & 0 & 8.33 \end{bmatrix} \text{ (kN/m)}^{-1}$$

Bar (3) : Redundant $\Rightarrow F_3 = F_3 = X = N_3$.

$$T^* = \begin{matrix} 1^* \\ 2^* \\ 3^* \end{matrix} \begin{matrix} F_1=1 & F_2=1 & F_3=1 \\ \hline 1.25 & 0 & 1 \\ -0.9375 & +1 & -1.2 \\ 0 & 0 & 1 \end{matrix}$$



$$N_1 \times \frac{3}{5} + N_2 = 0$$

$$f = \begin{matrix} T & P & T \\ F & & F \end{matrix} f^* \begin{matrix} T \\ F \end{matrix}$$

$$= \begin{bmatrix} 1.741 & \frac{-4.69}{10} & 1.603 \\ \frac{-4.69}{10} & \frac{5 \times 10^{-6}}{10} & -640 \\ 1.603 & \frac{-6}{10} & 2.386 \end{bmatrix} \times 10^{-3} \quad (\text{kN/m})^{-1}$$

Compatibility, :

$$f(F - F_{f^*}) = \{D\} - \{D_{\text{initial}}\} \quad F_{fA} = 0 \text{ here.}$$

$$D = D_{\text{initial}} + \{f\} \{F\}$$

$$\begin{Bmatrix} D_A \\ D_x \end{Bmatrix} = \begin{Bmatrix} D_{\text{initial}, A} \\ D_{\text{initial}, x} \end{Bmatrix} + \begin{bmatrix} f_{AA} & f_{Ax} \\ f_{xA} & f_{xx} \end{bmatrix} \begin{Bmatrix} 50 \\ -25 \\ X \end{Bmatrix}$$

$$\begin{Bmatrix} D_A \\ D_x \end{Bmatrix} = \begin{Bmatrix} 0 \\ -5 \text{ mm} \\ 0 \end{Bmatrix} + \{f\} \{F\} \quad \left(D_{\text{initial}} = \bar{T}_F^T D_{\text{initial}, x} \right)$$

Conjugate gradient Principle.

= -5 mm at (2)

$$D_A = \begin{Bmatrix} 0 \\ -5 \text{ mm} \end{Bmatrix} + f_{AA} \begin{Bmatrix} 50 \\ -25 \end{Bmatrix} + f_{Ax} X$$

$$D_x = 0 = f_{xA} \begin{Bmatrix} 50 \\ -25 \end{Bmatrix} + f_{xx} X$$

$D_x = 0$ for continuity of bar.
(assuming cut in bar)

$$\Rightarrow 2.386 X + (1.603 \times 50 + 25 \times \frac{6}{10}) = 0$$

$$\begin{aligned}
 D_A &= \begin{Bmatrix} 0 \\ -5 \text{ mm} \end{Bmatrix} + \begin{Bmatrix} 50 \\ -25 \end{Bmatrix} \cdot -40 \begin{Bmatrix} 1.603 \\ -0.6 \end{Bmatrix} \times 10^{-3} \\
 &= \begin{Bmatrix} 0 \\ -5 \end{Bmatrix} \times 10^3 + \left(\begin{Bmatrix} 1.741 \times 50 + \frac{4.69}{10} \times 25 \\ + 50 \times \frac{1.69}{10} - 25 \times \frac{5}{10} \end{Bmatrix} \right) \times 10^{-3} \begin{Bmatrix} 40 \times 1.603 \\ -40 \times 0.6 \end{Bmatrix} \\
 &= \begin{Bmatrix} 0 \\ -5 \end{Bmatrix} + \begin{Bmatrix} 98.775 \text{ mm} \\ -35.95 \text{ mm} \end{Bmatrix} - \begin{Bmatrix} 64.12 \text{ mm} \\ -24 \text{ mm} \end{Bmatrix}
 \end{aligned}$$

$$D_A = \begin{Bmatrix} 34.655 \text{ mm} \\ -16.95 \text{ mm} \end{Bmatrix}$$

Ans.

$$F^* = T_F \begin{Bmatrix} F \end{Bmatrix} = \begin{Bmatrix} 1.25 \times 50 + 1 \times (-40) \\ -0.9375 \times 50 + 1 \times (-25) - 1.2 \times (-40) \\ -40 \end{Bmatrix}$$

$$F^* = \begin{Bmatrix} 22.5 \\ -23.875 \\ -40 \end{Bmatrix} \text{ kN}$$

Ans.

(The slight change in the answers is due to rounding)